



## REVIEW ARTICLE

# OPTIMAL HARDENABILITY STEEL FOR ANY SIZE AND FORM OF MACHINE COMPONENTS TO INCREASE THEIR SERVICE LIFE AND DECREASE ALLOY ELEMENTS IN MATERIAL

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### ABSTRACT

In the paper an alloyed low hardenability steel to be used for any size and form of steel components is considered which is patented in Ukraine and can be used for providing an optimal hardened layer in product after its intensive quenching. This steel is often called as an optimal hardenability steel which provides high surface compressive residual stresses smoothly passing into smaller tensile stresses at the core of steel components. A fundamental correlation is proposed to compose chemical composition of optimal hardenability steel depending on size and form of product and grains of material's micro – structure. By painstaking experiments, it was established that during quenching often cracks occur if chemical composition of steel is not tolerant to size and form of a product. The paper also discusses the nature of extra surface hardness (3 – 6 HRC extra units) generated by compressive residual stresses on the surface of steel parts. It is underlined the importance of establishing correlation between compressive residual stresses and extra surface hardness that can be used in the future for controlling the correctness of designed chemical composition of steel which depends on size and configuration of quenched product. A method for simulation quenching processes of large steel components in lab condition is proposed that allows correct composing chemical composition of steel depending on dimension of product, form and grain size of material's micro – structure.

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## INTRODUCTION

In 1980 – 1983 of last century an idea was discussed on possibility to predict stress distribution in large steel components on the basis of simulation hardening processes in small samples (Morhuniuk, *et. al.*, 1982; Kobasko and Morhuniuk, 1983, 1985). That was very important for experimental simulation hardening processes of large machine components like rotors, big rollers and so on in lab condition. A powerful spray system was build (see Fig. 1) to quench steel components in condition of  $B_{iv} \rightarrow \infty$  or  $Kn = 1$ . Here  $B_{iv}$  is generalized Biot number and  $Kn$  is dimensionless Kondratjev number. The spray system occupied two levels in the experimental building. On the basement a powerful pump was arranged with a large water tank, flow meters and receiver. When developing intensive quenching process for KrAZ truck semi – axles, using powerful spray system (Fig. 1), an idea on optimal hardenability steel was developed in 1980 (Kobasko, 1980). The semi-axles of AutoKRAZ trucks were made of AISI 4340 steel and quenched in mineral oils resulting in rather big distortion. It was established that intensively quenched semi – axles made of AISI 1040 steel work much better as compared with semi - axles made of 4340 steel quenched in oil. Later were made numerous computer calculations concerning intensive quenching of different steel parts (Kobasko and Morhuniuk, 1985; Kobasko, Morhuniuk, *et.al.*, 1990). This paper summarizes the results of investigations and discusses possibility of use alloy low hardenability steel for large steel components of different forms and sizes.

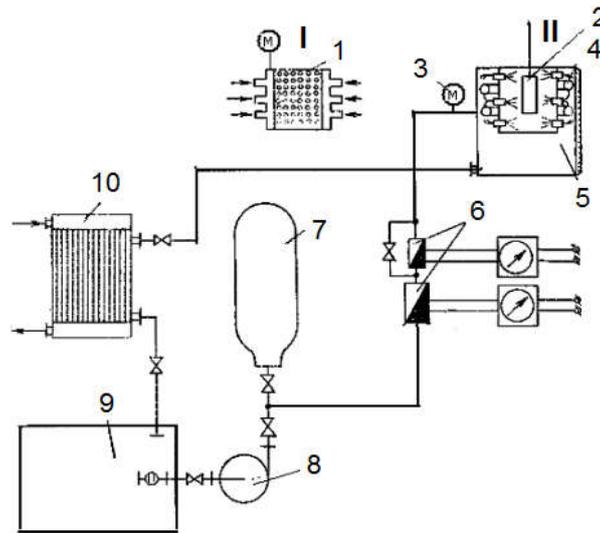
### An idea on alloy low hardenability steel for semi – axles of truck KrAZ raised by experiments

#### Intensive quenching of splined cylindrical samples

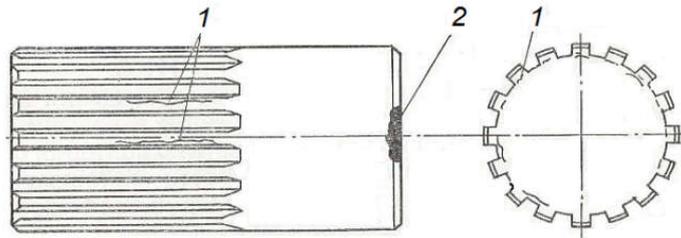
At the beginning the spray system shown in Fig. 1 was used for quenching cylindrical samples simulating hardening process of semi – axles.

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**Fig. 1. Installation for quenching steel parts and cylindrical specimens in water jets (Kobasko, 1980): I, removable sprayer with round holes made in the inner cylinder; II, water – air sprayer; 2, specimen; 3, manometer; 4, water – spray system; 5, tank with spray system; 6, equipment for measuring the amount of water used; 7, receiver; 8, pump; 9, water tank; 10, chiller**



**Fig. 2. Cylindrical splined samples 62 mm in diameter and 124 mm long made of different steels to be intensively quenched in spray system: 1, cracks in splines; 2, cracks at the edge of a sample**

Samples were made from different steel grades chemical composition of which is shown in Table 1. For each steel grade were made 6 samples. The samples were prepared by AutoKrAZ Co. for quenching them in lab condition in the frame of signed agreement. The goal of investigations was preventing any crack formation during intensive quenching, receiving maximal surface hardness, decreasing distortion of semi – axles, increasing their wear resistance, optimal cooling time correction and possibility of experimental simulation of quenching large steel parts in the lab condition. Removable size of sprayer 1 was manufactured depending on size and form of tested samples. For semi – axles samples inner diameter of sprayer was 120 mm and its height was 150 mm. A manometer was located on the top of a sprayer to measure pressure in it during experiments. Powerful pump provided pressure in a sprayer up to 1.6 MPa (Kobasko, 1980). Experiments were made in condition when pressure in sprayer was between 0.9 MPa – 1.6 MPa. Some results of testing are presented in Table 2.

**Table 1 Chemical composition of steels used for cylindrical samples simulating semi-axles of truck KrAZ to be tested in spray system shown in Fig. 1.**

Steel	C	Si	Mn	Cr	Ni	Cu	S	P	Notes
1040 GOST 1050	0.37 – 0.45	0.17 – 0.37	0.50 – 0.80	0.25 max	0.30 max	-	0.04 max	0.035max	
1050 GOST 1050	0.47 – 0.55	0.17 – 0.37	0.50 – 0.80	0.25 max	0.30 max	-	0.045 max	0.04 max	
47GT	0.44-0.51	0.10-0.25	0.95-1.25	0.25	0.25	0.30	< 0.04	< 0.04	Ti=0.06-0.012
4340	0.38 – 0.43	0.20 – 0.35	0.60 – 0.80	0.70-0.90	1.65 – 2.00	-	0.04 max	0.035max	Mo = 0.20- 0.30

Note that average content of carbon C in steel 1040 and 4340 is 0.41 wt % while average content of carbon in steel 47GT is 0.475 wt. % and 0.51 wt % in steel 1050 (see Table 1) that is almost 25 % larger..

**Table 2 Effect of intensive cooling time interruption on number of cracks and surface hardness after samples were quenched in sprayer system at a pressure 0.9 MPa.**

Steel	Cooling time, sec	Number of cracks	Hardness HRC on the top of spline	Hardness HRC on the cylindrical surface
47GT	10	5	49	42
	15	6	50	47
	20	10	57	50
4340	10	0	49	47
	15	0	52	49
	25	0	57	57
	40	0	61	65

Cylindrical splined samples (Fig. 2) were quenched in condition when Kondratjev number  $Kn = 1$  and extremely intensive quenching was interrupted at 10, 15, 20, 25 and 40 seconds to see how cooling time affect crack formation and surface hardness due to self – tempering (Table 2). Carefully conducted experiments showed that after intensive quenching of splined cylindrical specimens made of 1040 and 4340 steel there were no cracks at all while specimens made of 47GT cracked in all experiments (see Table 2 and Table 3). It is explained by higher content of carbon in 1050 and 47GT steels. These carefully conducted experiments in 1980 raised an idea on the importance of developing optimal hardenability steel depending on sizes and configurations of products and method of its optimization. Also, it was shown that intensive quenching generates extra surface hardness (3 – 6 HRC units). On the top of spline hardness was below normal value due to tensile residual stresses (Kobasko, Morhuniuk, *et.al.*, 1990) and hardness on cylindrical surface was above normal value due to compressive residual stresses on the surface of steel parts. It is very important to establish correlation between compressive residual stresses and extra surface hardness that can be used in the future for controlling the correctness of designed chemical composition of steel which depends on size and configuration of quenched product.

**Table 3. Number of splines with cracks vs. pressure for 1050 and 47GT steels after 40 sec of intensive cooling**

Steel	Pressure in sprayer, MPa	Number of cracks in 500 hrs	Type of crack
1050	0.9	4	2
	1.1	4	2
	1.4	1	1
47GT	0.1	6	2
	1.1	6	2
	1.3	2	2
	1.6	0	2

Table 3 shows number of splines with cracks in each specimen. Maximum number of cracks appears when all splines of sample are cracked during intensive quenching. For example, number 4 in Table 3 means presence quenching cracks in four splines in cylindrical sample. It should be noted here that quench cracks were observed only in samples made of 47GT and 1050 steel and, as was already mentioned, no cracks were observed after intensive quenching 1040 and 4340 steel. Table 4 shows core temperature of cylindrical sample immediately after intensive cooling interruption and average temperature of self – tempering process which was calculated using well known method (Kobasko, Aronov, *et. al.*, 2010).

**Table 4. Core temperature versus time of cooling for cylindrical splined samples**

Core temperature, °C	Cooling time, sec	Average temperature, °C
700	20	360
600	25	310
500	31	260
400	38	210
380	39	200
370	40	195

In spite of rather high temperature in the core of samples, cracks appeared in splines of cylindrical samples made of 1050 and 47GT steel. Due to very intensive quenching, splines were quenched through even after 10 sec of cooling. It means that probability of quench crack formation is higher in splined cylindrical samples made of 1050 and 47GT steel because specific volume of martensite is larger in 1050 and 47GT steel due to higher content of carbon (approximately 25%). During through hardening core swells to larger extent in steel containing more carbon. Core swelling results in tensile stresses which can generate quench cracks. Using HART – TANDEM software, authors (Kobasko, Morhuniuk, *et. al.*, 1990) calculated surface splines tensile stresses which were formed during intensive quenching of cylindrical samples shown in Fig. 2. These experiments and calculations showed that semi – axles can be made from 1040 steel of GOST 1050 (see Table 1). Further explanations concerning this approach are provided below.

#### Analysis of experimental intensive quenching processes

To calculate water stream at the outlet of nozzle, a well known Eq. (1) was used which follows from the Bernoulli's equations (Joseph, *et. al.*, 1978):

$$\Delta P = \eta \frac{\rho v^2}{2} \quad (1)$$

Where  $\Delta P$  is a pressure in sprayer,  $\eta$  is a coefficient depending on Re number and the nozzle configuration,  $\rho$  is liquid density,  $v$  is outlet velocity. Coefficient  $\eta$  is used for flooded streams. Eq. (1) can be rewritten as:

$$v = \sqrt{\frac{2\Delta P}{\rho\eta}} \quad (2)$$

Outlet velocity of water for single nozzle when stream is not flooded is provided in Table 3.

**Table 5. Outlet velocity of water from single nozzle versus pressure in sprayer shown in Fig. 1.**

P, Mpa	0.2	0.4	0.6	0.8	0.9	1.0	1.4	1.5	1.6
v, m/s	20	28	35	40	42	45	53	55	56

Average HTC's for impingement jet cooling can be calculated with accuracy of ± 15 % using the following dimensionless correlation (Martin, 1990).

$$\bar{Nu} = K_1 \cdot K_2 \cdot Re^{\frac{2}{3}} \cdot Pr^{0.42} \tag{3}$$

Where

$$K_1 = \left[ 1 + \left( \frac{H/D}{0.6} \sqrt{f} \right)^6 \right]^{-0.05}; \quad K_2 = \frac{\sqrt{f}(1 - 2.2\sqrt{f})}{1 + 0.2(H/D - 6)\sqrt{f}}; \quad f = \frac{(\pi/4)D^2}{A_{square(hexagon)}}$$

*D* is nozzle diameter;

*H* is a distance from the nozzle to the surface;

*A* is the area of the square or hexagon (see Fig. 7).

*f* is a relative nozzle area in terms of geometric dimensions of nozzle and cooled surface arrangements (for example, this parameter is equal to 0.25 · (D/r)<sup>2</sup> for the single round nozzle).

Dimensionless numbers *K*<sub>1</sub> and *K*<sub>2</sub> depend on the nozzle geometry and arrangement. Reynolds number *Re* depends on the outlet quenchant velocity and the number *Pr* characterizing physical properties of the quenchant. The above dimensionless correlation (3) is true for the following conditions: 2000 ≤ *Re* ≤ 100000; 0.004 ≤ *f* ≤ 0.04; 2 ≤  $\frac{H}{D}$  ≤ 12.

The task was to determine convective heat transfer coefficients for the sprayer described above in Fig. 1. First of all we will calculate all input parameters. *D*=0.002m; *H*=0.01m;

$$f = \frac{(\pi/4)D^2}{1 \cdot 10^{-4}} = 0.0314, \quad H/D = \frac{0.01m}{0.002m} = 5; \quad Pr = 7.03; \quad Pr^{0.42} = 2.268; \quad Re = \frac{10m/s \cdot 0.002m}{1.006 \cdot 10^{-6}m^2/s} = 19881; \quad Re^{\frac{2}{3}} = 734.$$

Now calculate dimensionless complexes *K*<sub>1</sub> and *K*<sub>2</sub>:

$$K_1 = \left[ 1 + \left( \frac{5}{0.6} \sqrt{0.0314} \right)^6 \right]^{-0.05} = 0.886; \quad K_2 = \frac{\sqrt{0.0314}(1 - 2.2\sqrt{0.0314})}{1 + 0.2(5 - 6)\sqrt{0.0314}} = 0.11.$$

It follows that

$$\bar{Nu} = 0.886 \times 0.11 \times 734 \times 2.268 = 162$$

$$\text{Whence } \alpha_{conv} = \frac{162 \times 0.597W/m \cdot K}{0.002m} = 47790W/m^2K.$$

Some results of convective heat transfer coefficients (HTCs) calculations are presented in Table 6 depending on outlet velocity *v* of water in sprayer and distance *H* to sample.

**Table 6. Convective heat transfer coefficients (HTCs) developed in spray system depending on outlet velocity *v* of water in sprayer and distance *H* to sample**

Outlet velocity in m/s	H, m	H/D	HTC, W/m <sup>2</sup> K
10	0.010	5	47790
	0.015	7.5	46758
	0.020	10	45725
20	0.010	5	75815
	0.015	7.5	74178
	0.020	10	72540

Table 7 presents results of French's experiments made in 1928 – 1930 (French, 1930).

**Table 7. Time required for the surface of steel spheres of different sizes to cool to different temperatures when quenched from 875°C in 5% water solution of NaOH at 20°C agitated with 0.914 m/s (French, 1930)**

Size, Inches, (mm)	Time, sec							
	700°C	600°C	500°C	400°C	300°C	250°C	200°C	150°C
4.75" (120.6)	0.043	0.066	0.09	0.12	0.17	0.21	0.29	0.95
7.15" (181.6)	0.040	0.070	0.100	0.140	0.240	0.310	0.42	1.15
11.25" (285.8)	0.043	0.120	0.190	0.330	0.570	0.960	1.26	2.18

As seen from Table 7, surface temperature of spherical steel samples 120.6 mm in diameter drops from 875°C to 300°C only for 0.17 sec. This time of cooling for large spherical steel part 285.8 mm in diameter was 0.57 sec. So fast cooling is explained by absence of any film boiling and it can be explained by creation of double electrical layer between metal surface and boundary liquid layer which prevents film boiling development due to crucial electrical forces taking place during quenching in electrolytes. Proceedings from fulfilled calculations and experiments, one can claim that cylindrical specimens shown in Fig.2 were quenched in sprayer ideally providing for them condition  $Kn = 1$ .

### Alloyed low hardenability steel and method of its designing

The powerful spray system shown in Fig. 1 can be successfully used for simulation hardening processes of large steel components in lab condition. The problem become very simple if boundary conditions are ideal when  $Bi = \infty$  and  $Kn = 1$ . In this case residual stresses in small model and big real steel part, like rotor or large roller, will be the same if are true Eqs.(4) (Kobasko, *et al.*, 1983, 1985):

$$\frac{a\tau}{R^2} = idem; \quad \frac{r}{R} = idem; \quad \frac{DI_a}{D_{opt}} = idem. \quad (4)$$

Here  $DI_a$  is critical size for real steel components which has the same configuration as real steel part has;  $D_{opt}$  is size of real steel component which is manufactured from optimal hardenability steel (patented alloy low hardenability steel, UA Patent No. 114174, C2). The meaning of idem is "the same".

The proposed alloy low hardenability steel contains following alloy elements in wt.% (UA Patent No. 114174):

C:	0.30 - 1.20
Mn:	≤ 0.20
Si:	≤ 0.20
Cr:	≤ 0.50
Ni:	≤ 1.60
Mo:	≤ 0.25
Cu:	≤ 0.20
Al:	0.03 - 0.10
Ti:	0.05 - 0.12
V:	≤ 0.40
S:	≤ 0.035
P:	≤ 0.035
Fe:	bal,

It differs from early known low hardenability steels, which are shown in Table 8 and Table 9, by containing additionally molybdenum and elevated amount of nickel, up to 1.6 wt.%.

**Table 8. Chemical composition of low hardenability (LH) steels used by authors (Shepelyakovskii and Ushakov, 1990; Ouchakov, 1998)**

Steel	C	Si	Mn	Cr	Ni	Cu	S	P	Notes
58 (55PP)	0.55-0.63	0.10-0.30	0.20	0.15	0.25	0.20	0.04	0.04	$\Sigma(Mn+Cr+Ni) \leq 0.5$
ShKh4	0.95-1.05	0.15-0.30	0.15-0.30	0.35-0.50	0.30	0.25	0.027	0.020	$\Sigma(Cu+Ni) \leq 0.5$
45C	0.42-0.48	0.40-0.65	0.17-0.32	0.25	0.20	0.15	0.040	0.035	
55C	0.53-0.60	0.40-0.80	0.21-0.35	0.25	0.20	0.20	0.040	0.04	
70PP	0.66-0.73	0.15-0.30	0.15-0.30	0.25	0.25	0.25	0.040	0.04	
115PP	1.10-1.20	0.15-0.30	0.40-0.60	0.25	0.20	0.20	0.040	0.040	Ti=0.06-0.12
ShKh2	1.15-1.25	0.15-0.30	0.15-0.30	0.15	0.10	0.12	0.027	0.020	Al=0.015-0.030; $\sigma \leq 0.030$

As seen from Table 9, critical diameter of existing LH steel is rather small (only 8 – 13 mm) and it means that it is suitable only for small and medium machine components. New alloy LH steel can be used for different forms and larger sizes of steel components. Low hardenability (LH) steel (Shepelyakovskii and Bezmenov, 1998; RU Patent № 2158320, 1999) increases service life of machine components when correctly used for intensively quenched small and medium gears, shafts and other configurations and cannot be used for large components like large wind gears, rotors, big rollers, and so on.

Table 9. Chemical composition of low hardenability (LH) steels according to RU Patent № 2158320

Элементы та DI	Chemical composition, wt %					
	62PP <sub>1</sub>	62PP <sub>2</sub>	62PP <sub>3</sub>	62PP <sub>4</sub>	62PP <sub>5</sub>	80PP
C	0.60-0.67	0.60-0.67	0.60-0.67	0.60-0.67	0.60-0.67	0.78-0.085
Mn	0.05-0.15	≤0.10	0.05-0.15	0.10-0.20	≤0.06	≤0.10
Si	≤0.05	0.10-0.20	0.05-0.15	0.10-0.20	≤0.06	≤0.05
Cr	≤0.10	≤0.10	≤0.10	≤0.10	≤0.06	≤0.10
Ni	≤0.10	≤0.10	≤0.10	≤0.10	≤0.06	≤0.10
Cu	≤0.10	≤0.10	≤0.10	≤0.10	≤0.06	≤0.10
Al	0.03-0.10	0.03-0.10	0.03-0.10	0.03-0.10	0.03-0.10	0.03-0.10
Ti	0.06-0.12	0.06-0.12	0.06-0.12	0.06-0.12	0.06-0.12	0.06-0.12
V	≤0.4	≤0.4	≤0.4	≤0.4	0.2-0.3	≤0.04
S	≤0.04	≤0.04	≤0.04	≤0.04	≤0.04	≤0.04
P	≤0.035	≤0.035	≤0.035	≤0.035	≤0.035	≤0.035
DI, mm	8-13	8-12	8-13	11.5-15.5	8.5-9.5	10-12

Recently issued, a new patent on alloy LH steel (UA Patent № 114174, 2017) solves this problem making possibility use it for large steel components on the basis of developed method of its composing (Kobasko, 2017). The proposed method is based on fundamental correlation (5) which has a form:

$$\frac{DI_a}{D_{opt}} = 0.35 \pm 0.095. \quad (5)$$

Where

$$DI_a = f(DI, Form). \quad (6)$$

According to Grossmann (Grossmann, 1964) critical diameter DI for cylinder depends on chemical composition of steel, Eq. (7), and it can be calculated as:

$$DI = 25.4 \times f_{Fe} \times f_{Mn} \times f_{Si} \times f_{Cr} \times f_{Ni} \times \dots \quad (7)$$

Hardenability factors  $f_{Fe}$  and  $f_x$  are provided by Table 10. For cylindrical forms  $DI_a = DI$ . For more complicated forms of steel parts function (6) should be used which is calculated by computer program. This program can also find suitable chemical composition among already existing steel grades to fit for any form and size of product  $D_{opt}$  to be optimized.

Table 10. Hardenability factors  $f_{Fe}$  and  $f_x$  for steel depending on grain size and selected alloying elements (Totten *et.al.*, 1993)

Content of carbon, wt. %	$f_{Fe}$ depending on grain size and content of carbon			$f_x$ depending on content of alloy elements, wt. %				
	No. 6	No. 7	No. 8	Mn	Si	Ni	Cr	Mo
0.10	0.1153	0.1065	0.0995	1.333	1.070	1.036	1.2160	1.30
0.20	0.1623	0.1509	0.1400	1.667	1.140	1.073	1.4320	1.60
0.30	0.1991	0.1849	0.1700	2.000	1.210	1.109	1.6480	1.90
0.35	0.2154	0.2000	0.1842	2.167	1.245	1.128	1.7560	2.05
0.40	0.2300	0.2130	0.1976	2.333	1.280	1.146	1.8640	2.20
0.45	0.2440	0.2259	0.2090	2.500	1.315	1.164	1.9720	2.35
0.50	0.2580	0.2380	0.2200	2.667	1.350	1.182	2.0800	2.50

To be sure that 1040 steel of GOST 1050 provides optimal hardened layer and is optimal for semi – axles 62 mm in diameter, let's make calculation using correlation (5). Semi- axles for field testing were made from steel chemical composition of which was: 0.40C; 0.60 Mn; 0.30 Si; 0.10 Cr; 0.10 Ni; in wt. %. Knowing chemical composition of 1040 steel, one can calculate  $DI_a$  for cylinder using data from Table 10. It is:

$$DI_a = 25.4 \times 0.198 \times 3.00 \times 1.21 \times 1.216 \times 1.036 = 23. mm$$

And

$$\frac{DI_a}{D_{opt}} = \frac{23}{62} = 0.37 = 0.35 + 0.02$$

that perfectly satisfies Eq. (4).

As a result, fatigue tests of KrAZ truck semi – axles were excellent (see Table 11).

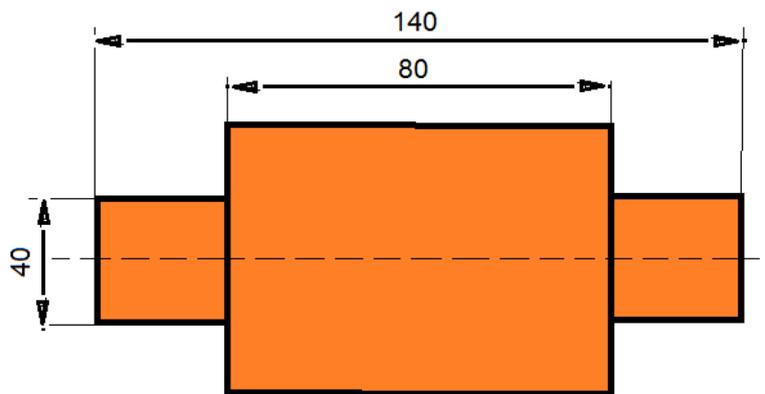
**Table 11. Fatigue tests of KrAZ truck semi – axles (Kobasko. 1980)**

Quenching method	Steel grade	Numbers of cycles to fracture	Notes
Oil	AISI/SAE 4340	$3.8 - 4.6 \times 10^5$	Semi – axles were destroyed
Intensive water spray cooling	AISI/SAE 1040	$3.0 - 3.5 \times 10^6$	No fracture observed

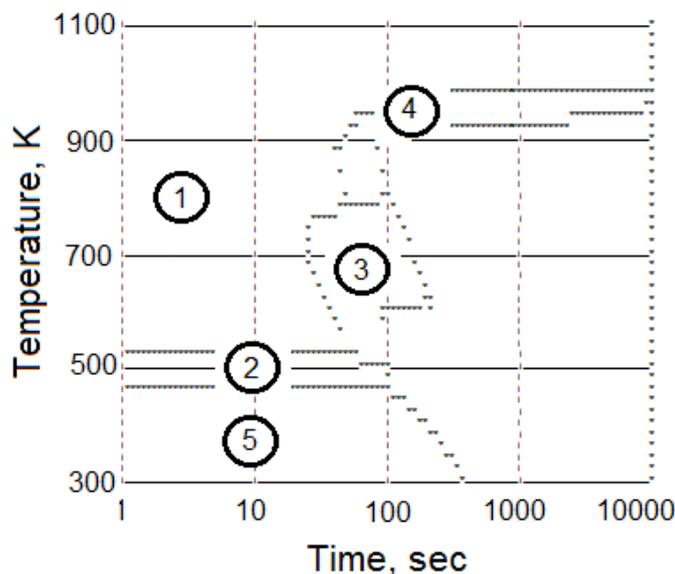
It turned out that conventional 1040 steel fits thickness of semi – axle 62 mm in diameter to be an optimal composition for this specific size. Since alloy low hardenability steel is designed for medium and large steel parts, it is extremely important to develop a method of experimental and computer simulation quenching processes of large steel components in lab condition because experiments with real large steel components like rotors are very expensive. This problem is discussed below. .

**Computer simulation of large steel parts quenching processes**

Assume that there is a roller 800 mm in diameter and 1400 mm long which should be manufactured from the alloy low hardenability steel. Its configuration is similar to model shown in Fig. 3, however, it is 10 times larger. Real large roller will be quenched intensively in condition when  $Kn = 1$  in water tank with a moving flooded sprayer (Kobasko, 2013). There is a need to investigate stress and phase distribution in large roller during its intensive quenching and evaluate safe factor to see whether quench cracks could appear during hardening. A small model of large roller is shown in Fig. 3 and is manufactured from the AISI/SAE 52100 steel which satisfies approximately fundamental Eq. (5). The model shown in Fig. 3 can be quenched in removable sprayer 1 of Fig. 1 in condition  $Kn = 1$  and investigated by FEM (finite element method) computer calculations if CCT (continuous curves transformation) diagram (Fig. 4) and mechanical properties of material vs. temperature are known. Such calculations and analysis related to small model shown in Fig. 3 are provided below.



**Fig. 3. Drawing of a roller (model) with the maximal diameter 80 mm and 140 mm long to simulate hardening processes in 10 times larger roller for which  $DI_a / D_{opt}$  is the same**



**Fig. 4. CCT diagram which provides high surface compressive residual stresses in small roller when intensively quenched; 1 is austenite; 2 is martensite; 3 is bainite; 4 is pearlite; 5 is martensite after quenching is complete**

Mechanical properties of steel shown in Fig. 4 vs. temperature can be find in literature (Kobasko, Morhuniuk, *et. al.*, 2017).

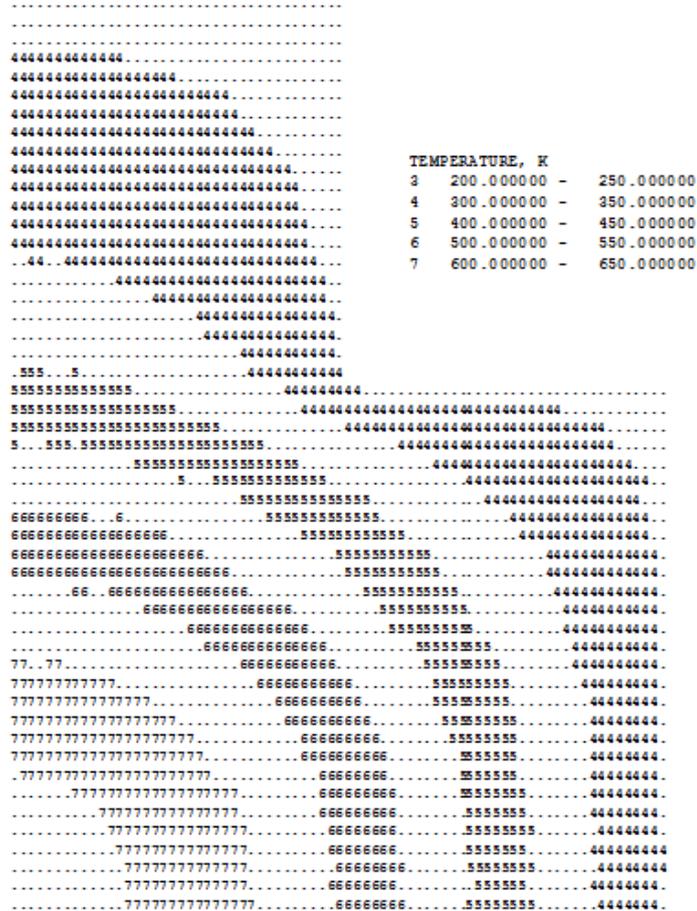


Fig. 5 Temperature field in a small roller in K at the moment of time70 sec when quenched in condition Kn = 1.

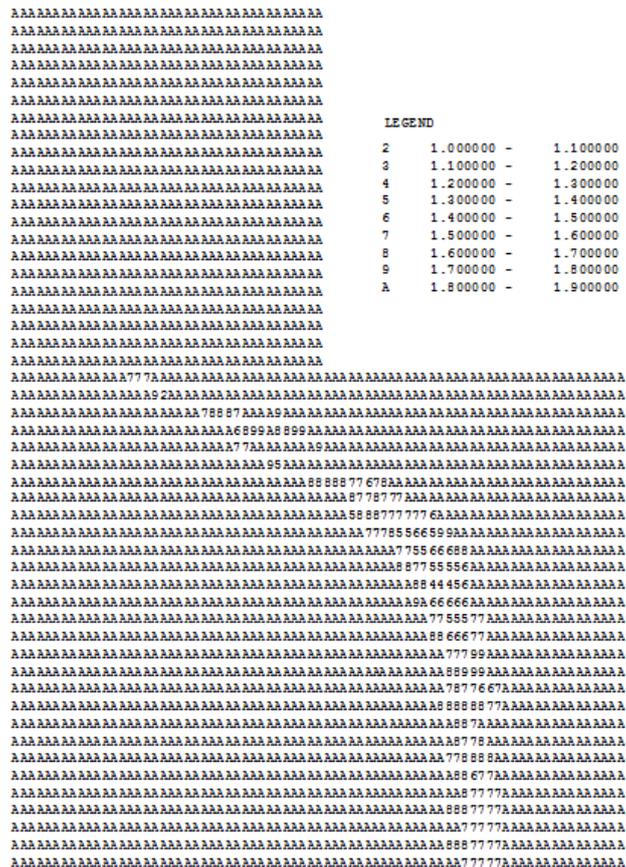


Fig. 6 Safe factor's  $k(x,y,\tau)$  field in a small roller at the moment of time70 sec when quenched in condition Kn = 1.

Fig. 6 supports an idea on preventing quench cracking when quenching is very intensive and uniform. Combining this important fact with the optimized chemical composition of steel, one can guarantee absence of crack formation in large steel components like rotors and big rollers. More information on safe factor  $k(x,y,\tau)$  one can find in Ref. (Pisarenko and Lebedev, 1976; Morhuniuk, 1982).



Fig. 7 Micro – structure distribution in a small roller at the moment of time 70 sec when quenched in condition Kn = 1: 3 is bainite, 4 is pearlite, 5 is martensite.

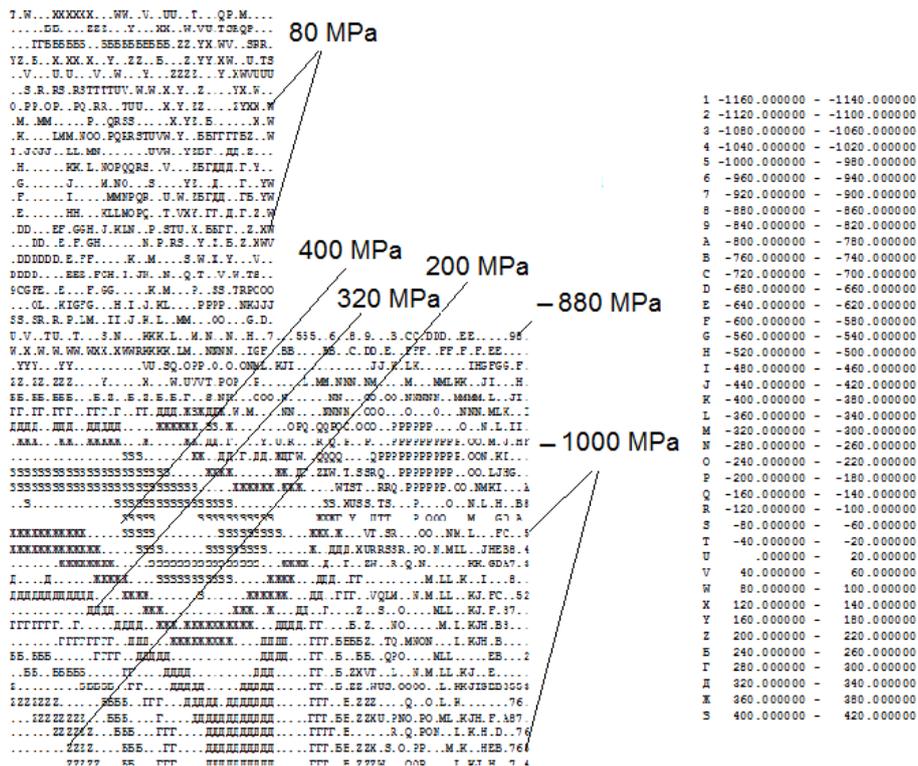


Fig. 8 Hoop stress field in a small roller in MPa at the moment of time 70 sec when quenched in condition Kn = 1.

As seen from Fig. 6 and Fig. 7, the lowest safe factor  $k$  and the boundary between bainitic and martensitic structure have the same location. It means that probability of crack formation is high along the boundary bainitic – martensitic structure. Results of computer calculations, presented in Fig. 8, support the main rule according to which on the surface of through hardened steel parts neutral or tensile stresses during quenching are formed. In our case, in through hardened cylindrical section of the roller surface hoop stresses were 80 MPa (see Fig. 8). Surface compressive hoop stresses in cylinder with the section of 180 mm were more than – 1000 MPa (see Fig. 8). A ratio of hardened layer (100% martensite) to large diameter of the is equal to 0.27. Core of the roller has pearlitic structure and bainitic structure is located between pearlite and martensite.

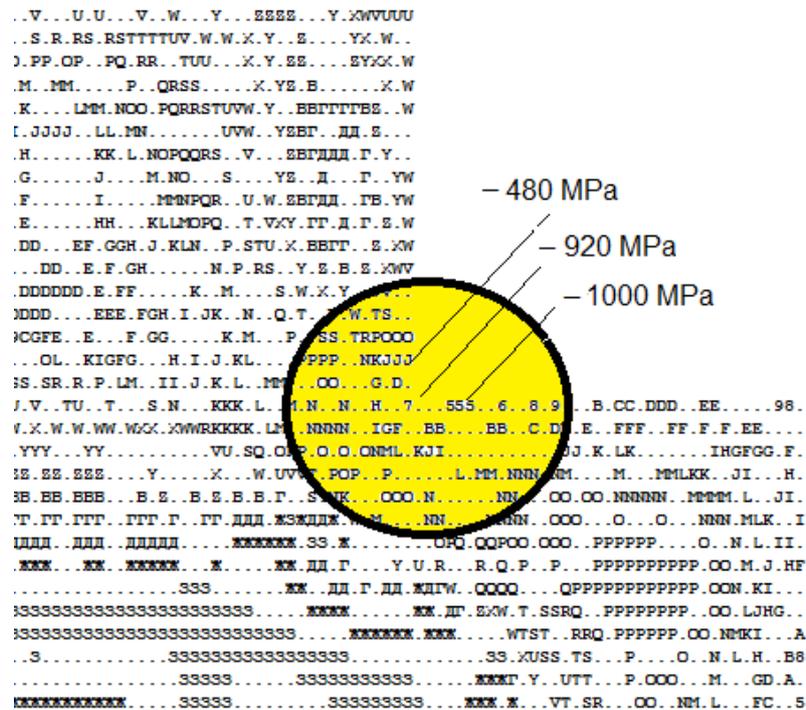


Fig. 9. Surface hoop stresses in the intersection of different diameters of small roller in MPa at the moment of time 70 sec when quenched in condition  $K_n = 1$ .

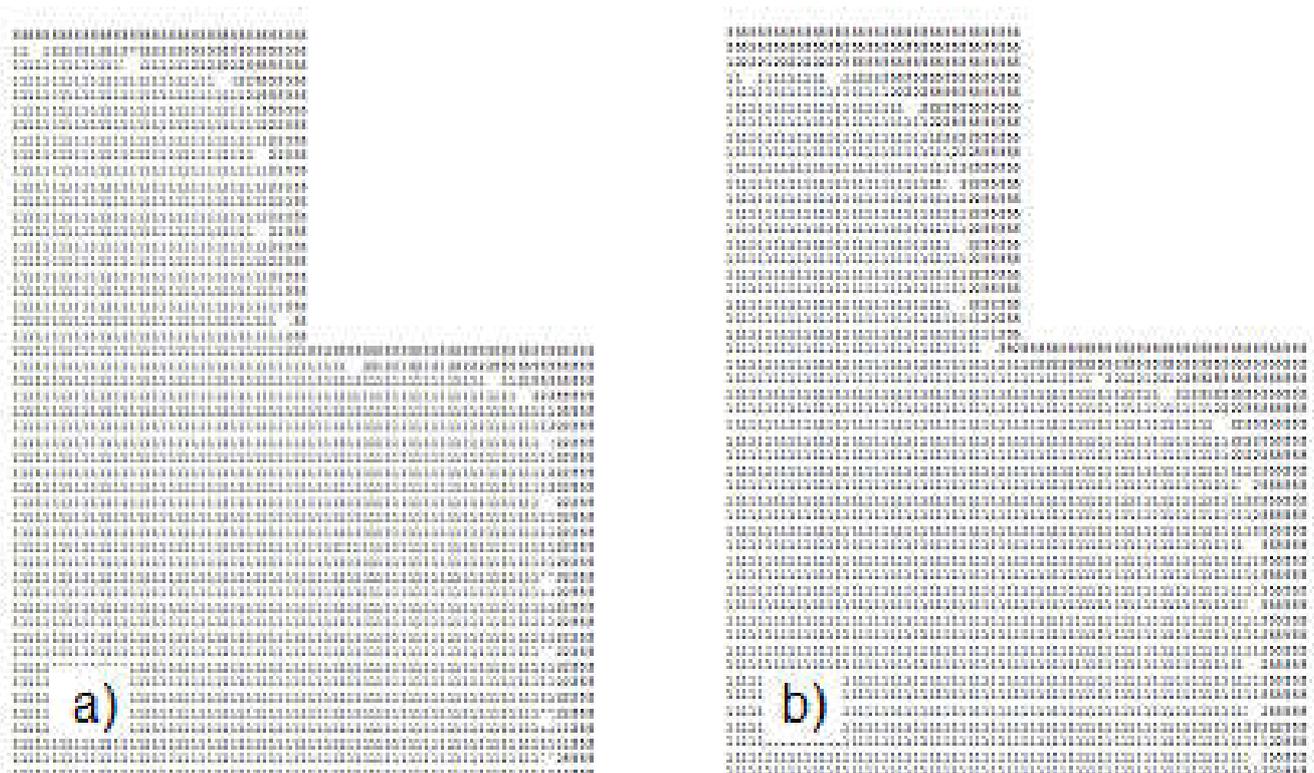


Fig. 10. Formation of martensitic surface shell in small roller at the beginning of cooling in condition  $K_n = 1$  : a), after 5 sec; b), after 10 sec; 1 is austenite, 5 is martensite

As follows from Fig. 9, at the interseccio of different diameters of roller, high surface compressive hoop stresses are formed . It is due to formation of martensitic shell which is smoothly distributed around the surface of a roller (see Fig. 10). In area pont 7, where small diameter of the roller appraches the large diameter, compressive hoop stresses are: - 480 MPa, - 920 MPa, -1000 MPa (see Fig. 9). It is clear to everybody that such comprerssive stress distribution cannot generate quench crack formation. In point 7 save factor k is highest and is equal to 1.9. Quench cracks are formed when  $k(x,y,\tau) \leq 1$  (Pisarenko – Lebedev, 1976). Fig. 10 shows the martensitic shell formed around the roller at the moment of intensive cooling 5 sec (a) and 10 sec (b).

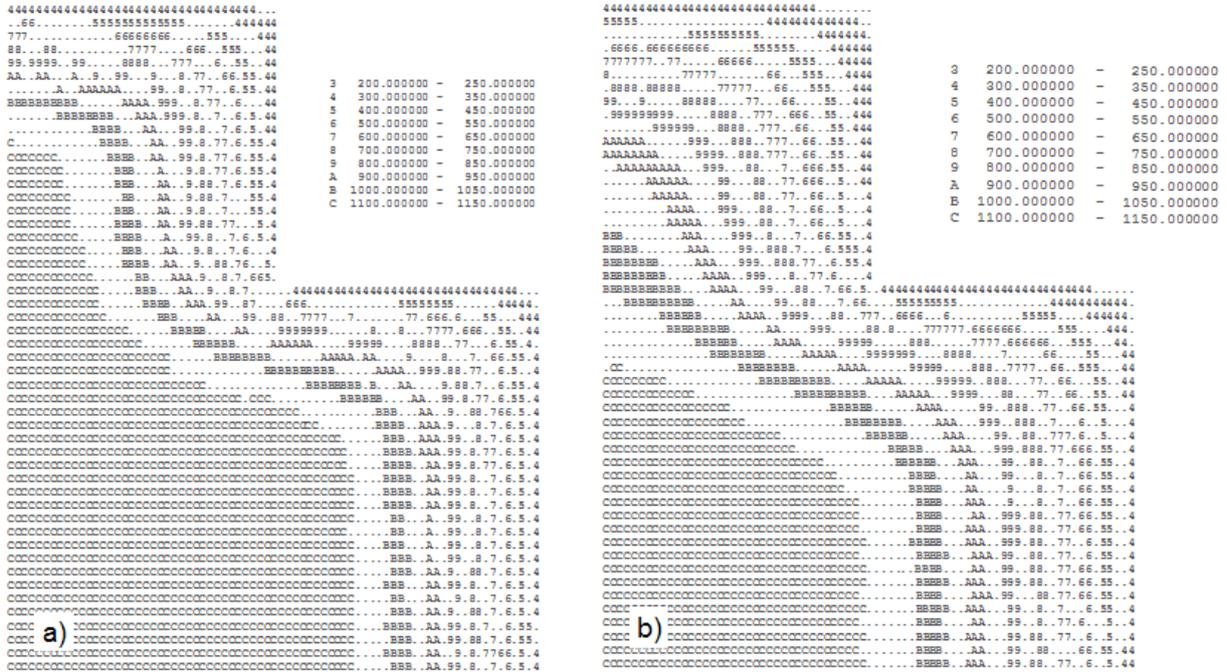


Fig. 11 Temperature field in K in small roller at the beginning of cooling in condition Kn = 1 : a), after 5 sec; b), after 10 sec.

Fig. 11 presents the temperature field in Kelvin centigrade (K) in the small roller at the beginning of cooling in condition Kn = 1 at the moment of time 5 sec (a) and 10 sec (b). At the beginning of cooling the martensitic shell is covered hot core of the roller consisting of plastic austenite. No quench cracks in such condition are possible. Thus, computer simulation shows that no quench crack at all when quenching rollers, made of optimal hardenability steel, in ideal condition provided by flooded movable sprayers

(Kobasko, 2013). Similar stress and phase distribution will be in 10 times larger roller if  $\frac{\alpha\tau}{R^2}$ ,  $\frac{r}{R}$ , and  $\frac{DI_a}{D_{opt}}$  are the same.

**DISCUSSION**

There are two companies dealing with intensive quenching processes to suggest their developments to interested customers: IQ Technologies Inc. (IQT) and Intensive Technologies Ltd. (ITL). IQ Technologies Inc is an engineering consulting firm founded in 1999 in Akron, Ohio, USA. IQ Technologies Inc is dedicated to enhancing the performance, environmental soundness, cost effectiveness, and safety of heat-treating processes. The prime its mission is to commercialize intensive quenching processes for steel parts in US and abroad and to provide intelligent solution of wide range of heat-treating problems. It is committed to IntensiQuench® process and equipment engineering excellence, and it is working as a team consisting of DANTE Solution Inc., Air Flow Science Corporation, AFC-Holcroft, Ajax TOCCO Magnethermic, etc. The consulting company Intensive Technologies Ltd (ITL) was founded in 2000 in Kyiv, Ukraine and its prime mission is development of new intensive quenching processes, designing appropriate software for governing of developed new technologies. It is cooperating with leading experts from National Academy of Sciences of Ukraine and National Metallurgical Academy of Ukraine. Combined activity of both companies will bring a great benefit to heat treating industry because IQT mainly focuses on designing and manufacturing equipment for IQ processes while ITL focuses on academic painstaking experimental and analytical investigations to govern correctly technological processes. As a good example for illustrating a great sense of mutual cooperation is an alloy low hardenability steel, discussed in this paper, which will be used sooner or later worldwide for IQ – 3 processes including large steel parts like rotors and others (UA Patent No. 109577, 2015). Accurate computer simulation can be done for them using contemporary computer codes (Dowling, Pattok, Ferguson *et al.* 1996; Inoue and Arimoto, 1997; Ferguson, *et al.*, 2002, 2007).

**Conclusion**

- Alloy low hardenability steel containing molybdenum and elevated amount of nickel is proposed to be used for any size and form of machine components to increase their service life and decrease alloy elements in material when intensively quenched. The alloyed low hardenability steel has been patented in Ukraine (Ukrainian Patent UA 114174, C2) and is currently subjected to its further careful investigations. .

- A method for composing alloyed low hardenability steel depending on size and configuration of machine component is proposed which is based on regular condition theory and theory of similarity.
- A possibility of simulation process of intensive quenching very large components made of alloyed low hardenability steel in a lab experimental condition is realistic if Fourier number, dimensionless coordinates, and  $DI_a/D$  are the same.
- Intensively quenched alloy low hardenability steel provides high surface compressive residual stresses which exceeds 1000 MPa and relatively low tensile stresses at the core of steel parts.
- High surface compressive residual stresses are the reason for high surface hardness which can be HRC 65 for AISI 1040 steel.
- Different hardness on cylindrical and splined surfaces is explained by differences in residual surface stresses.
- Optimal hardenability steel for specific steel part can be designed as a new one or chosen from already existing grades using fundamental correlation for steel chemical composition optimization.

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